# Evaluation of $\boldsymbol{K}$-Factor of Preloaded Bolted Joints 

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#### Abstract

Load-bearing capacity and resistance of preloaded joints bolts depends on the value of torque. Correlation between tightening torque and axial preloading force is made by means of $k$-factor, whose value depends on the mechanical characteristics of the components of bolted structural joints. In this work is presented a way that experimental evaluation factor.


Key words: preloaded bolted joints, high-strength preloaded bolts

## Introduction

The design and checking calculation of steel structures is carried out in accordance with the standards of Eurocod 3. There are 25 standards governing the design of steel structures.
EN 1993-1-8 [1] refers to the joints using high strength preloaded bolts because much of the metal constructions are carried out using bolted joints with high-strength preloaded bolts.
Transmission of loads between the bolt (figure 1) -and components for steel construction which combines - is done by frictional forces generated between the areas of application of these contact elements, within the limits of friction forces, capable of mounting bolts to pre-stressing.


Fig. 1. Preloaded bolt joints.
Since the maximum tasks undertaken by such seams will depend on the: number of bolts; mechanical and physical characteristics of the clamped members, in particular the bolts; mechanical characteristics of surfaces in contact, and in particular the degree of processing of these surfaces, cover them with other substances, corrosion etc.; the bolts are made of alloy steel with higher resistance obtained by appropriate heat treatment and contact surfaces of the joints shall be processed in order to achieve a coefficient of friction as possible.
According to [1] for joints with high strength preloaded bolts use only screws of 8.8 and 10.9 the first number is the ratio $f_{u b} / 100$, and the second report $10 \cdot f_{y b} / f_{u b}$, the $f_{y b}\left(=R_{p 0,2}\right)$ is the
yield strength of the steel, and $f_{u b}\left(=R_{m}^{\min }\right)$ of the steel of bolt. All according to [1] denominations to yield strength $f_{y b}$ and minimum tensile strength $f_{u b}$ are given in table 1 .

Table 1. Denominations for the yield strength $f_{y b}$
and minimum tensile strength $f_{u b}$ of high strength preloaded bolts.

| Bolt class | 8.8 | 10.9 |
| :---: | :---: | :---: |
| $f_{v b}(\mathrm{MPa})$ | 640 | 900 |
| $f_{u b}(\mathrm{MPa})$ | 800 | 1000 |

The design slip resistance of preloaded bolts should be taken as [1]:

$$
\begin{equation*}
F_{s, R d}=\frac{k_{s} n \mu}{\gamma_{M 3}} F_{p, C} \tag{1}
\end{equation*}
$$

where: $k_{s}$ is a factor what consider the deviation of contour of hole, given in Table $2 ; n$ is the number of friction surfaces; $\mu$ is the slip factor obtained either by specific tests for the friction surface in accordance with EN 1090-2 [2], Annex G or when relevant as given in Table 3 [1]; $\gamma_{\mathrm{M} 3}=1.25$ is partial safety factor for joint for slip resistance al ultimate limit state, given in Table 2 [1]; $F_{p, C}$ is preloading force.

Table 2. Values of $k_{s}[1]$.

| Description | $k_{s}$ |
| :--- | :--- |
| Bolts in normal holes | 1.0 |
| Bolts in either oversized holes or short slotted holes with the axis of the slot perpendicular to the <br> direction of load transfer | 0.85 |
| Bolts in long slotted holes with the axis of the slot perpendicular to the direction of load transfer | 0.7 |
| Bolts in short slotted with the axis of the slot parallel to the direction of load transfer | 0.76 |
| Bolts in long slotted with the axis of the slot parallel to the direction of load transfer | 0.63 |

Table 3. Slip factor, $\mu$, for pre-load bolts [1].

| Surface treatment | Class | $\mu$ |
| :--- | :--- | :--- |
| Surface blasted with shot or grit with loose rust removed, not pitted | A | 0.50 |
| Surface blasted with shot or grit: <br> a) spray-metalized with a aluminium or zinc based product; <br> b) with alkali-zinc silicate paint with a thickness of $50 \mu \mathrm{~m}$ to $80 \mu \mathrm{~m}$ | B | 0.40 |
| Surface cleaned by wire-brushing or flame cleaning, with loose rust removed | C | 0.30 |
| Surface as rolled | D | 0.20 |

In accordance with [2], the nominal minimum preloading force $F_{p, C}$ shall be taken as:

$$
\begin{equation*}
F_{p, C}=0,7 \cdot f_{u b} \cdot A_{s} \tag{2}
\end{equation*}
$$

where: $f_{u b}$ is the nominal ultimate strength of the bolt material; $A_{s}$ is the stress area of the bolt.
As you can see strength of preloaded bolt joints depends on preloading force. Accordingly implementation of preloaded bolts (the axial load) should be carried out more specifically. For this purpose use wrenches calibrated and verified by the competent laboratories. Correlation between the torque $M_{s}$ and the tension created is described by the relationship:

$$
\begin{equation*}
M_{s}=k \cdot d \cdot F_{p, C} \tag{3}
\end{equation*}
$$

where $d$ is bolt diameter and $k$ - nut factor, sometimes called the friction factor.
Value of the coefficient $k$ is given by the manufacturer of bolts and is determined experimentally. According to $k$-factor are the three classes of set bolt-nut, shown in table 3 . User agent of preloaded high strength bolts is required to check the value of the coefficient $k$.

According to the relationship (3) bolt tension generated during tightening depends on the accuracy and value of the torque and the coefficient $k$. To ensure you can use either pre-stressed of tightening method provided in table 4 , unless specified restrictions on their use.

Table 3. $K$-classes

| $k$ - class | Information to be supplied |
| :--- | :--- |
| K0 | No requirements for $k$-factor |
| K1 | Range of individual test value $k_{i}$ |
| K2 | Mean test value $k_{m}$ <br> Coefficient of variation of $k$-factor $V_{k}$ |

Table 4. $K$-classes for tightening methods

| Tightening | $k$-classes |
| :--- | :--- |
| Torque method | K2 |
| Combined method | K2 or K1 |
| HRC tightening method | K0 with HRD nut only or K2 |
| Direct tension indicator (DTI) method | K2, K1 or K0 |



Fig. 2. Test set-up:
1 - Nut: turned during tightening; 2 - Washer of the assembly: prevented from rotating: 3-Shim(s); 4-Calibrated bolt force measuring device; 5 -
Chamfered washer of the assembly or chamfered shim; 6 -Bolt head: prevented from rotating; $a$ - Clamp length.

Before mounting bolts, should be carried out calibration. This calibration is done by making a test to determine the necessary parameters to ensure that the minimum required pre-stressed is produced in a safe way by tightening methods indicated in [2, 3]. The test consists of the collection and measurement of agent during the gathering, bolt tension generated during tightening and torque.

For test purposes you can use a device for measuring the force in bolt according to [3], or mechanical or hydraulic equipment such as a load transducer. The torque indicator handle wrench used for the test must be those used on the site. They must have an appropriate area of application.

Separate tests must be carried out on representative samples taken from each batch of the joint involved. The test items should be selected so that all relevant aspects of their conditions must be identical. The representative must consist of a number of bolts, nuts and washers from each batch. The test set-up (see Figure 2) may include shims needed to suit the measuring device.
The test assemblies and shims shall be positioned such that:

- a washer of the assembly is placed under the nut;
- a chamfered washer or a chamfered shim is placed under the bolt head;
- the clamp length including shims and washer(s) is the minimum allowed in the relevant product standard.


## Experimental Details

Examinations were carried out on high strength bolts of mechanical characteristics of the group, with the following s 10.9: M24 x 80; M24 x 115. They were placed at the disposal of the beneficiary, in each dimension have tried the 8 screws. Manufactured by the mechanical characteristics are indicated: the moment of tightening $M_{s}=800 \mathrm{Nm}$; the value of the factor $k=0.1 \ldots 0.16$; preloading force $F_{p, C}=220 \mathrm{kN}$. High strength bolts tested does not require the use of lubricants.

Determination of preloading force was made by direct tension indicator (DTI) method. Tests were performed on the devices which have clamp length specified for each dimension of bolt joint.
For tightening, have used the handle wrench, which enabled the reading the torque, and which are used to tightening bolts in the construction site. For the evaluation of preloading force used a resistive force transducer (force washer) produced by HBM, with call sign KMR 400, shown in Figure 3. Transducer used has the following main features: 400 kN nominal force; deviation from linearity: $10 \%$; the sensitivity of $1.7 \ldots 2 \mathrm{mV} / \mathrm{V}$. This resistive transducer has been connected to a data acquisition system ESAM Traveller Static, which was connected to a laptop.

Resistive force transducer KMR was mounted on each of the bolts, as shown in Figure 1, with the clamping length for each size of $a$ bolt. After installation, using the handle wrench and load on each assembly bolt-nut, applied the following torques: $M_{s}=400 \mathrm{Nm} ; 550 \mathrm{Nm} ; 700 \mathrm{Nm} ; 800 \mathrm{Nm}$ and 900 Nm . For each value of the moment to raise there has been preloading force in bolt.

## Results

In tables 5 and 6 values obtained for the two sizes of bolts are shown.
Based on the data in tables 5 and 6 was calculated with the value of the coefficient $k$ (3) relationship (tab. 5 and 6).

Table 5. The results of measurements

| $\begin{aligned} & M_{s}, \\ & \mathrm{Nm} \end{aligned}$ | $\begin{gathered} \hline \text { Bolt } 1 \\ \text { M24x } 80 \\ \hline \end{gathered}$ |  | $\begin{gathered} \text { Bolt } 2 \\ \text { M24×80 } \\ \hline \end{gathered}$ |  | $\begin{gathered} \hline \text { Bolt } 3 \\ \text { M24x80 } \end{gathered}$ |  | $\begin{gathered} \hline \text { Bolt } 4 \\ \text { M24x } 80 \\ \hline \end{gathered}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $F_{p, C}, \mathrm{kN}$ | $k$ | $F_{p, C}, \mathrm{kN}$ | $k$ | $F_{p, C}, \mathrm{kN}$ | $k$ | $F_{p, C, \mathrm{kN}}$ | $k$ |
| 450 | 115.4 | 0.143 | 128.5 | 0.146 | 104 | 0.180 | 117.56 | 0.159 |
| 550 | 138.9 | 0.146 | 155.4 | 0.147 | 135 | 0.170 | 141.92 | 0.161 |
| 700 | 202.5 | 0.147 | 196.5 | 0.148 | 178.14 | 0.164 | 184.04 | 0.158 |
| 800 | 228.5 | 0.149 | 246.2 | 0.135 | 215.43 | 0.155 | 213.63 | 0.156 |
| 900 | 251.1 | 0.153 | 275.2 | 0.136 | 243.94 | 0.154 | 246.04 | 0.152 |
| $M_{s},$ | Bolt 5M24x 80 |  | $\begin{gathered} \hline \text { Bolt } 6 \\ \text { M24×80 } \\ \hline \end{gathered}$ |  | $\begin{gathered} \hline \text { Bolt } 7 \\ \text { M24×80 } \\ \hline \end{gathered}$ |  | $\begin{gathered} \text { Bolt } 8 \\ \text { M24×80 } \\ \hline \end{gathered}$ |  |
|  | $F_{p, C, \mathrm{kN}}$ | $k$ | $F_{p, C, \mathrm{kN}}$ | $k$ | $F_{p, C, \mathrm{kN}}$ | $k$ | $F_{p, C, \mathrm{kN}}$ | $k$ |
| 450 | 140.51 | 0.133 | 124.7 | 0.150 | 120.29 | 0.156 | 119.39 | 0.157 |
| 550 | 164.92 | 0.139 | 148.21 | 0.155 | 141.87 | 0.161 | 121.58 | 0.188 |
| 700 | 214.72 | 0.136 | 190.54 | 0.153 | 181.16 | 0.161 | 166.92 | 0.175 |
| 800 | 247.96 | 0.134 | 226.04 | 0.147 | 213.49 | 0.156 | 205.95 | 0.162 |
| 900 | 294.95 | 0.127 | 263.53 | 0.142 | 241.53 | 0.155 | 251.43 | 0.149 |

Table 6. The results of measurements

| $\begin{aligned} & M_{s}, \\ & \mathrm{Nm} \end{aligned}$ | $\begin{gathered} \text { Bolt } 1 \\ \text { M } 24 \times 115 \end{gathered}$ |  | $\begin{gathered} \hline \text { Bolt } 2 \\ \text { M } 24 \times 115 \end{gathered}$ |  | $\begin{gathered} \hline \text { Bolt 3 } \\ \text { M24x115 } \end{gathered}$ |  | $\begin{gathered} \text { Bolt } 4 \\ \text { M } 24 \times 115 \end{gathered}$ |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | $F_{p, C, \mathrm{kN}}$ | $k$ | $F_{p, C, \mathrm{kN}}$ | $k$ | $F_{p, C, \mathrm{kN}}$ | $k$ | $F_{p, C, \mathrm{kN}}$ | $k$ |
| 450 | 118.5 | 0.158 | 116.04 | 0.162 | 111.04 | 0.169 | 120.7 | 0.155 |
| 550 | 143.5 | 0.160 | 138.5 | 0.165 | 136.5 | 0.168 | 131.4 | 0.174 |
| 700 | 205.3 | 0.142 | 207 | 0.141 | 188.3 | 0.155 | 185.5 | 0.157 |
| 800 | 230.2 | 0.145 | 222.6 | 0.150 | 220.5 | 0.151 | 218.9 | 0.152 |
| 900 | 253 | 0.148 | 279.2 | 0.134 | 256.9 | 0.146 | 238.5 | 0.157 |
| $M_{s}$ | $\begin{gathered} \hline \text { Bolt } 5 \\ \mathrm{M} 24 \times 115 \end{gathered}$ |  | $\begin{gathered} \text { Bolt } 6 \\ \text { M24x115 } \\ \hline \end{gathered}$ |  | $\begin{gathered} \hline \text { Bolt } 7 \\ \text { M } 24 \times 115 \end{gathered}$ |  | $\begin{gathered} \text { Bolt } 8 \\ \text { M } 24 \times 115 \end{gathered}$ |  |
|  | $F_{p, C, \mathrm{kN}}$ | $k$ | $F_{p, C, \mathrm{kN}}$ | $k$ | $F_{p, C,}$ kN | $k$ | $F_{p, C, \mathrm{kN}}$ | $k$ |
| 450 | 108.12 | 0.173 | 115 | 0.163 | 117.9 | 0.159 | 106.6 | 0.176 |
| 550 | 141.5 | 0.162 | 130.2 | 0.176 | 128 | 0.179 | 140.8 | 0.163 |
| 700 | 181.6 | 0.161 | 183.2 | 0.159 | 177.2 | 0.164 | 180.6 | 0.161 |
| 800 | 215.8 | 0.154 | 212.6 | 0.157 | 210 | 0.159 | 238.3 | 0.140 |
| 900 | 249.3 | 0.150 | 240.9 | 0.156 | 240.6 | 0.156 | 267 | 0.140 |

According to [2,4], for the two strings values of coefficient $k$, commensurate with the size and M 24 x 80 and M 24 x 115 , respectively determine the arithmetic mean $k_{M}$ and standard deviation $s$. These calculated values are given in table 7.

Table 7. Statistical data processing

| Bolt | $k_{M}$ | $s$ | $0,08 k_{M}$ |
| :---: | :---: | :---: | :---: |
| M 24 x 80 | 0.152521 | 0.012162 | 0.012202 |
| M 24 x 115 | 0.15749 | 0.010628 | 0.012658 |

To validate the results, standard deviation shall not exceed $8 \%$ [4] of the average value of the coefficient, as can be seen from table 7 that condition is fulfilled by the two sizes of bolts.

## Conclusions

From the analysis of the values obtained and presented in table $5 \ldots 7$ recommendation a few conclusions:

- In general, values of $k$ are within the range prescribed by the manufacturer, namely between $0,1 \ldots 0,16$, any of the overruns interval can be explained by $10 \%$ deviation from linearity of the transmitter by force;
- The average values of the coefficient $k_{M}$ are within the range given by the manufacturer; For these values at some point tightening $M_{s}=800 \mathrm{Nm}$, shown by the manufacturer in bolts develops maximum preloading force $F_{p, C}=218 \mathrm{kN}$ which leads to a maximum stress $\sigma_{\max }=$ 617 MPa max. If the bolts are made of steel with $f_{u b}=900 \mathrm{MPa}$ then, tension $\sigma_{\max }$ is less than $0.7 f_{u b}=700 \mathrm{MPa}$ as required norm. For this reason it is recommended that the tightening is not less than $M_{s}=800 \mathrm{Nm}$ for bolts made of steel with $f_{u b}=100 \mathrm{MPa}$, in order to avoid overrunning the flow of the material where the coefficient $k$ would have at the beginning of the interval $0,1 \ldots 0,16$.


## References

1. ***-EN 1993-1-8 Eurocode 3: Design of steel structures; Part 1-8: Design of joints.
2.     *         *             *                 - EN 1090-2 Eurocode 1: Execution of steel structures and aluminium structures; Part 2. Technical requirements for steel structures
3. ***-EN 14399-2 High-strength structural bolting assemblies for preloading; Part 2: Suitability test for preloading;
4.     *         *             *                 - STAS 9330-84 Poduri de cale ferată şi şosea. Îmbinări cu şuruburi de înaltă rezistență. Prescripții de proiectare şi execuție.

## Evaluarea factorului $k$ al asamblărilor cu şuruburi de înaltă rezistență pretensionate

## Rezumat

Capacitatea portantă şi rezistența îmbinărilor prin şuruburi de inaltă rezistență pretensionate depinde de valoarea momentului de strângere. Corelarea dintre momentul de strângere şi efortul axial dezvoltat in şurub se face prin intermediul factorului $k$, a cărui valoare depinde de caracteristicile mecanice ale elementelor asamblării. În lucrare este prezentată o modalitate de evaluare experimentală a acestui factor.

